

IN THE CLAIMS:

1. **(Currently Amended)** A safety coupling arrangement comprising a coupling part that can be adapted for fixed co-action with a shaft[[,]] or axle ~~or the like~~, that functions to transfer torque and rotary movement to said safety coupling, and a further coupling part which is adapted for fixed co-action with a shaft[[,]] axle ~~or the like~~ to transfer torque and rotational movement from the safety coupling, and further including a safety unit, wherein said safety unit is adapted to take one of two settings, a first setting in which torque and rotary movement can be transferred between said two coupling parts and a second setting in which no torque and rotational movement can be transferred between said two coupling parts, wherein said safety unit includes a subpart or a body that can take said first setting as a result of an expansion caused by applying pressure to a cavity within the safety unit and enclosing said pressure in said cavity, and is able to take its second setting by evacuating said pressure from said cavity, wherein a first coupling part or a second coupling part includes an axially directed, or generally axially directed, groove, wherein said groove is adapted to be able to surround a pressure expandable subpart in said safety unit and the whole, or essentially the whole, of said cavity and wherein said expandable subpart, when in its first setting, functions to allow torque to be transferred directly to said first coupling part or said second coupling part via two mutually opposing

surface parts, which are frictionally active against opposing outer parts of the axially directed groove, wherein said first coupling part includes two axially-directed projections each adapted for coaction with a respective groove in the second coupling part, and are shorter than said cavity, wherein the expandable part is formed as an end locking means, and said locking means provides two axially-directed edges.

2. **(Previously Presented)** An arrangement according to Claim 1, wherein said one or said other coupling part includes a collar which is centred or essentially centred with respect to said axially directed groove.

3. **(Previously Presented)** An arrangement according to Claim 2, wherein said collar is integrated with said one coupling part or said other coupling part.

4. **(Previously Presented)** An arrangement according to Claim 3, including an outer radial groove formed between a flange and said collar, said flange belonging to one or the other coupling part.

5. **(Previously Presented)** An arrangement according to Claim 4, wherein a thin material section is formed in said one or said other coupling part between said outer radial groove and said axially directed groove of said safety unit.

6. **(Previously Presented)** An arrangement according to Claim 5, wherein said thin material section is elastically resilient.

7. **(Previously Presented)** An arrangement according to Claim 1, wherein said safety unit is integrated with and constitutes said first

coupling part or said second coupling part and includes a flange for fixed co-action with the torque transferring connected to the safety coupling.

8. **(Previously Presented)** An arrangement according to Claim 1, including a pressure medium filling nipple, such as oil, which extends radially out from the safety unit and is positioned in connection with said one coupling part or said other coupling part and its collar.

9. **(Previously Presented)** An arrangement according to Claim 8, including a device, which is fixed in relation to said collar, which co-acts with or is able to co-act with said filling nipple such that a small relative movement between said first coupling part and said second coupling part and said safety unit will cause the filling nipple to shear, for a rapid evacuation of said pressure.

10. **(Previously Presented)** An arrangement according to Claim 1, including a first ball bearing ring placed at the bottom of the safety unit groove for co-action between said safety unit and said groove.

11. **(Previously Presented)** An arrangement according to Claim 10, including a second ball bearing ring placed adjacent an opening of said safety unit groove for co-action between the safety unit and said groove.

12. **(Previously Presented)** An arrangement according to Claim 1, wherein said axially directed groove has a conical cross-sectional shape with the widest part facing towards an adjacent part.

13. **(Previously Presented)** An arrangement according to Claim 12, wherein the cross-sectional shape of said subpart and its sections has a corresponding conical shape.

14. **(Previously Presented)** An arrangement according to Claim 13, wherein said axially directed groove and said corresponding sections have a stepped cross-sectional shape with the widest part facing towards an adjacent part.

15. **(Previously Presented)** An arrangement according to Claim 1, wherein the free end portions of the material sections or the legs forming said axially directed groove are coordinated with locking means provided there between and adapted to prevent any divergence of said free end portions when the safety unit, together with its associated subpart or body, takes its first and expanding setting.

16. **(Previously Presented)** An arrangement according to Claim 15, wherein said first coupling part and said second coupling part are mutually adapted to include mutually overlapping and coordinated cylindrical subsections on a respective side of an axially directed groove .

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18. **(Currently Amended)** An arrangement according to Claim ~~17~~18, wherein said edges are related peripherally to said first coupling part, and wherein said groove is formed peripherally in said second coupling part.

19. **(Previously Presented)** An arrangement according to Claim 16, wherein said subsection is adapted for torque transmission via axially orientated and cylindrical outer sections.

20. **(Previously Presented)** An arrangement according to Claim 19, wherein the length of said outer sections and a normal pressure dependent on the chosen expansion of the expandable subpart are adapted for a torque transfer of between 10 and 30% of the total torque transferred between said coupling parts.

21. **(Previously Presented)** An arrangement according to Claim 20, wherein the chosen torque transfer is adapted to between 15 and 25%.

22. **(Previously Presented)** An arrangement according to Claim 20, wherein the axially directed groove has a length of more than 50% of the length of said expandable subpart.

23. **(Previously Presented)** An arrangement according to Claim 22, wherein said length is adapted to be less than 80% of the axial length of said expandable subpart.

24. **(Previously Presented)** An arrangement according to Claim 16, wherein said overlapping subsections have the same, or essentially the same, radial thicknesses.

25. **(Previously Presented)** An arrangement according to Claim 15, wherein with regard to the overlapping subsections, the outer subsection has a greater thickness than the inner.

26. **(Previously Presented)** An arrangement according to Claim 15, wherein the first coupling part is formed to function as a locking means against expansion of the free end portions of the legs forming said groove in the second coupling part.

27. **(Previously Presented)** An arrangement according to Claim 26, wherein the legs forming said groove have the same, or essentially the same, material thickness.

28. **(Previously Presented)** An arrangement according to Claim 27, wherein the radius difference between the mutually opposing cylindrical outer parts of the groove is smaller than, equal to or essentially equal to the total radial thickness of said free end portions or legs.